

## Contact Stress Calculation

### Input data

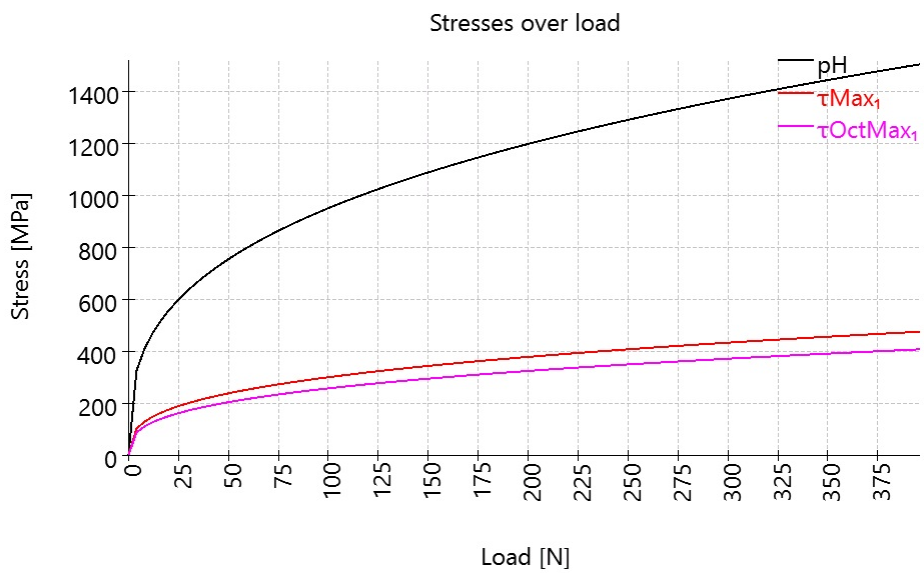
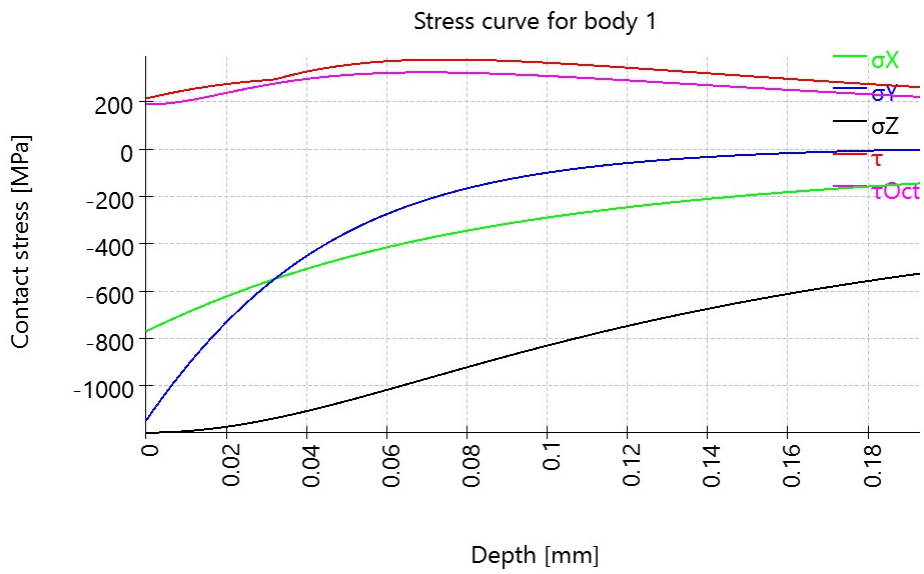
First radius body 1	$r_{11}$	5.0000 mm
Second radius body 1	$r_{12}$	5.0000 mm
First radius body 2	$r_{21}$	-5.2000 mm
Second radius body 2	$r_{22}$	100.000 mm
Effective length for cylinders	$l_{eff}$	0.0000 mm
Normal force	$F_n$	200.000 N
Youngs modulus body 1	$E_1$	210000 MPa
Youngs modulus body 2	$E_2$	210000 MPa
Poisson number body 1	$\nu_1$	0.3
Poisson number body 2	$\nu_2$	0.3
Angle between planes for radii	$\alpha$	0.0000 °

### Results

Major half axis of contact ellipse	$a$	0.8172 mm
Minor half axis of contact ellipse	$b$	0.0975 mm
Approach of both bodies	$\delta$	0.0036 mm
Contact stiffness	$R$	84259.6 N/mm
Hertzian stress	$\rho_H$	1198.6 MPa
Maximal orthogonal shear stress	$\tau_{xz}$	91.882 MPa
Depth for max. orthogonal shear stress	$z(\tau_{xz})$	0.0729 mm
Maximal orthogonal shear stress	$\tau_{yz}$	298.606 MPa
Depth for max. orthogonal shear stress	$z(\tau_{yz})$	0.0482 mm
Maximal shear stress body 1	$\tau_{Max_1}$	378.946 MPa
Maximal octahedral shear stress body 1	$\tau_{OctMax_1}$	324.941 MPa
Depth for max. shear stress body 1	$z(\tau_{Max_1})$	0.0747 mm
Depth for max. octahedral shear stress body 1	$z(\tau_{OctMax_1})$	0.0698 mm
Equivalent stress body 1 (Tresca)	$\sigma_{eTresca_1}$	757.893 MPa
Equivalent stress body 1 (Mises)	$\sigma_{eMises_1}$	689.305 MPa
Maximal shear stress body 2	$\tau_{Max_2}$	378.946 MPa
Maximal octahedral shear stress body 2	$\tau_{OctMax_2}$	324.941 MPa
Depth for max. shear stress body 2	$z(\tau_{Max_2})$	0.0747 mm
Depth for max. octahedral shear stress body 2	$z(\tau_{OctMax_2})$	0.0698 mm
Equivalent stress body 2 (Tresca)	$\sigma_{eTresca_2}$	757.893 MPa
Equivalent stress body 2 (Mises)	$\sigma_{eMises_2}$	689.305 MPa

# MESYS Shaft and Rolling Bearing Calculation

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